# **SOLAR COOLING**

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# Why Solar Cooling

- Dramatic increase of air conditioning since the early 80ies
- Cost of energy
- Issues related to environmental pollution
  - Due to energy production
  - Due to the use of CFC's and HCFC's
- Matches demand with source availability
- Crucial for improving life standards in developing countries



## **Thermal Comfort**

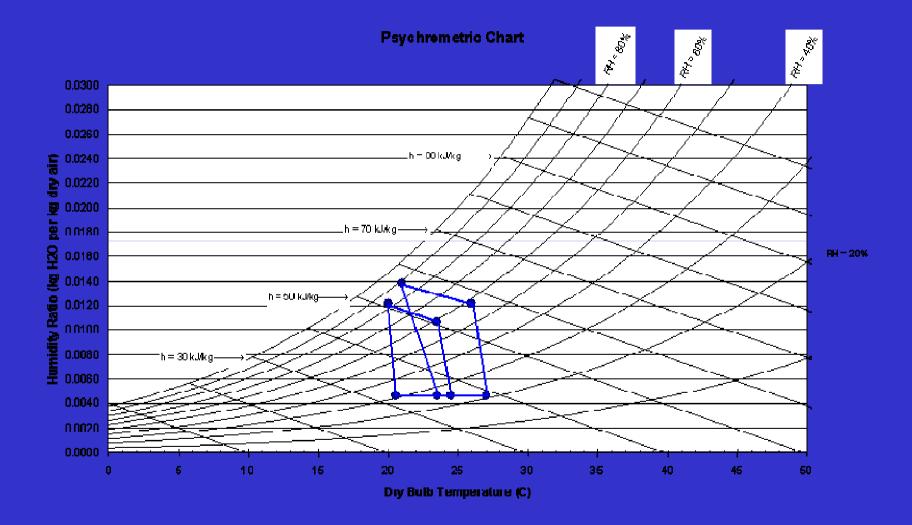
"Is that condition of mind that expresses satisfaction with the thermal environment"

Depends on may parameters: Meteorological Physiological / psychological Clothing etc

Conclusion: Concept **not easily** quantifiable!



#### Thermal Comfort – ASHRAE Approach





# **Underlying Physics**

Thermodynamics

1<sup>st</sup> Law: The change of internal energy ( $\Delta U$ ) of a system is equal to the heat absorbed (Q), plus the external work (W) done on the system

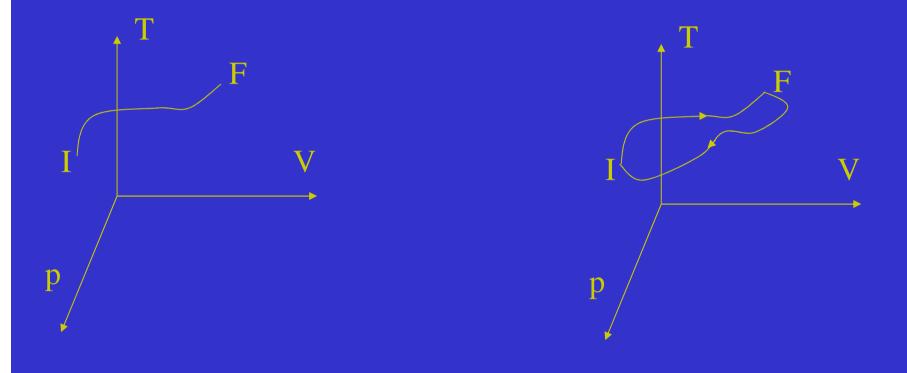
W, Q related to the changes the system experiences when going from an initial to a final state



# Thermodynamic Cycle

Simple Transformation

Cyclical Transformation or Cycle





# Entropy

The concept of entropy was originally introduced in 1865 by Rudolf Clausius. He defined the *change in entropy* of a thermodynamic system, during a reversible process in which an amount of heat  $\Delta Q_r$  is applied at constant absolute temperature *T*, as

#### $\Delta S = \Delta Qr / T$

Clausius gave the quantity *S* the name "entropy", from the Greek word  $\tau \rho \sigma \pi \dot{\eta}$ , "transformation". Since this definition involves only differences in entropy, the entropy itself is only defined up to an arbitrary additive constant



# Thermodynamics - 2<sup>nd</sup> Law

The most probable processes that can occur in an isolated system are those in which entropy increases or remains constant

In other words:

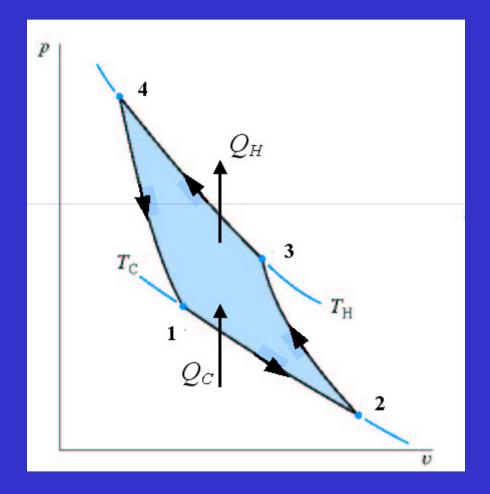
In an isolated system there is a well-defined trend of occurrence of process and this is determined by the direction in which entropy increases.

In other words:

Heat flows naturally from a system of higher temperature to a system of lower temperature.



## **Ideal Carnot Refrigeration Cycle**



1→2 Isothermal expansion
2→3 Adiabatic compression
3→4 Isothermal compression
4→1 Adiabatic expansion

 $W_{cycle} = \int_{1}^{2} P dv + \int_{2}^{3} P dv + \int_{3}^{4} P dv + \int_{4}^{1} P dv$ = shaded area (net work *in*)



#### Coefficient of Performance (COP)

Useful cooling energy

COP =

Net energy supplied by external sources



#### Latent Heat

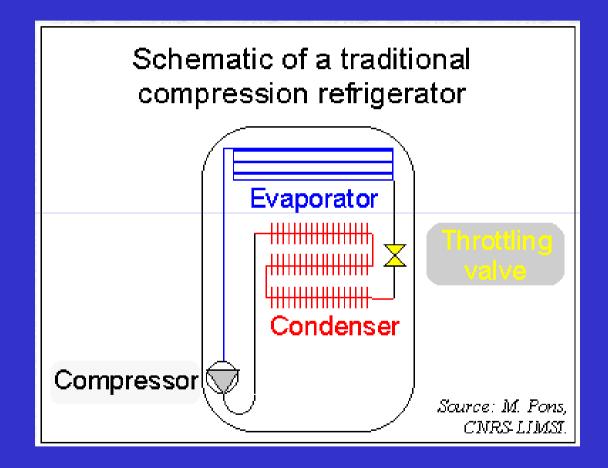
Is the amount of <u>energy</u> in the form of <u>heat</u> released or absorbed by a substance during a change of <u>phase</u> (i.e. solid, liquid, or gas), – also called a phase transition.<sup>[</sup>

Two latent heats are typically described: latent <u>heat of fusion (melting</u>), and latent <u>heat of vaporization (boiling</u>). The names describe the direction of heat flow from one phase to the next: solid  $\rightarrow$  liquid  $\rightarrow$  gas. The change is <u>endothermic</u>, i.e. the system absorbs energy, when the change is from solid to liquid to gas. It is <u>exothermic</u> (the process releases energy) when it is in the opposite direction.

Because energy is needed to overcome the molecular forces of attraction between water particles, the process of transition from a parcel of water to a parcel of vapor requires the input of energy causing a drop in temperature in its surroundings. If the water vapor condenses back to a liquid or solid <u>phase</u> onto a surface, the latent energy absorbed during evaporation is released as <u>sensible heat</u> onto the surface. The large value of the latent heat of condensation of water vapor is the reason that steam is a far more effective heating medium than boiling water, and is more

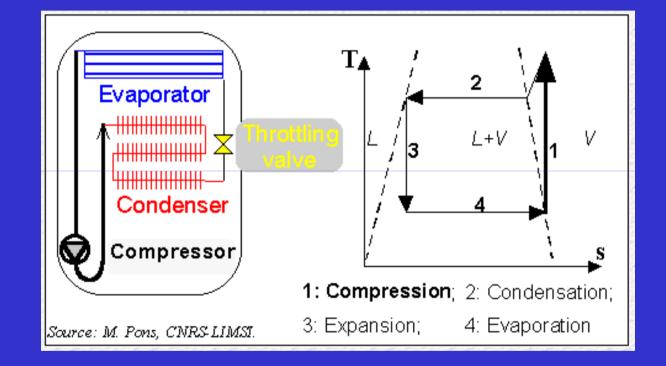


# Conventional cooling cycle





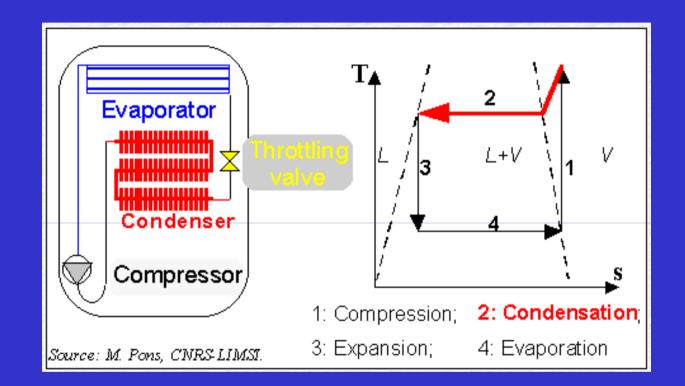
# Compression



# Vapor is compressed and its temperature increases (p V = n R T)



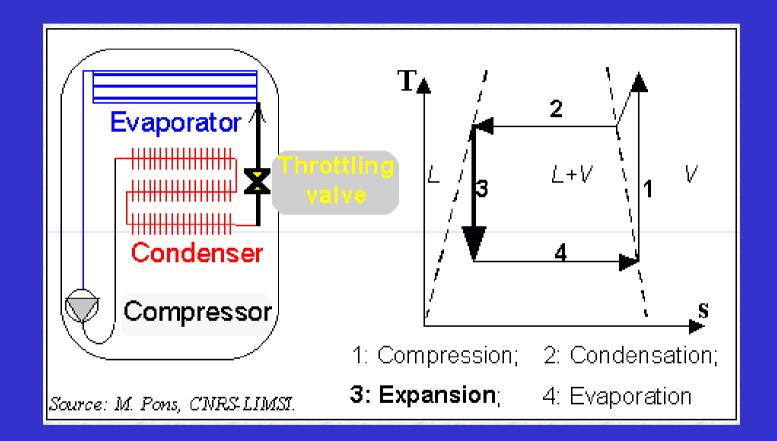
## Condensation



# The fluid at "high pressure" is cooled by ambient air and therefore condensed



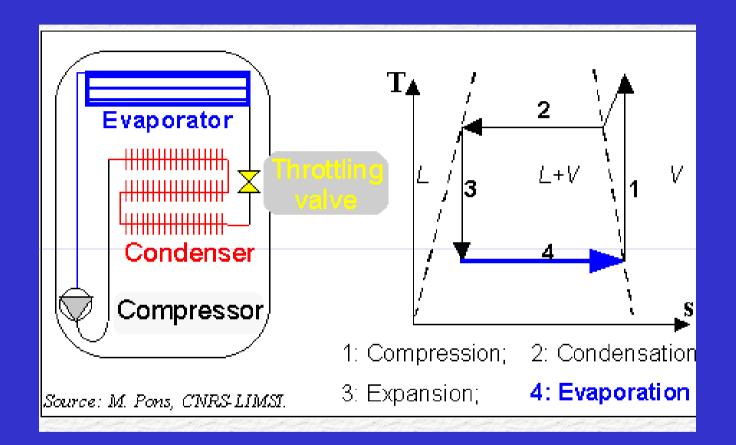
## Expansion



The liquid refrigerant is depressurized and its temperature decreases (p V = n R T)



## Evaporation



The liquid refrigerant at "low pressure" receives heat at low temperature and evaporates



#### **Thermal Solar Cooling Techniques**

Absorption Cooling

Energy is transferred through phase-change processes

Adsorption Cooling

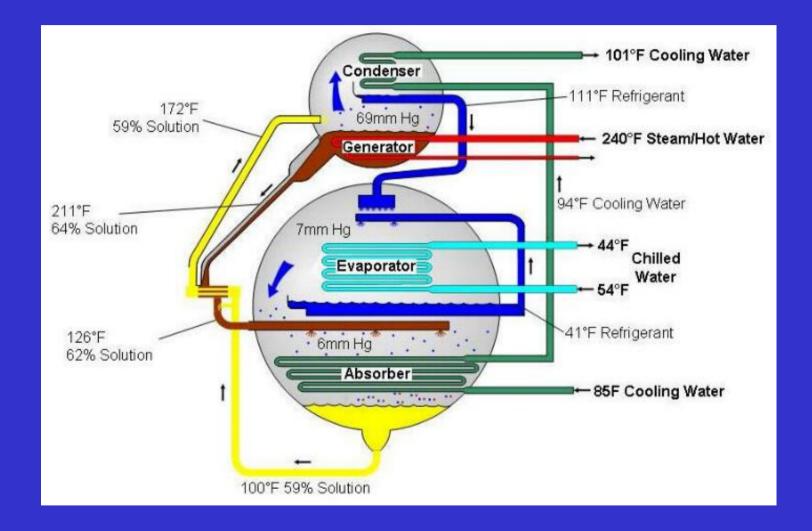
Energy is transferred through phase-change processes

**Desiccant Cooling** 

Energy is transferred through latent heat processes



# Absorption Cooling (1)





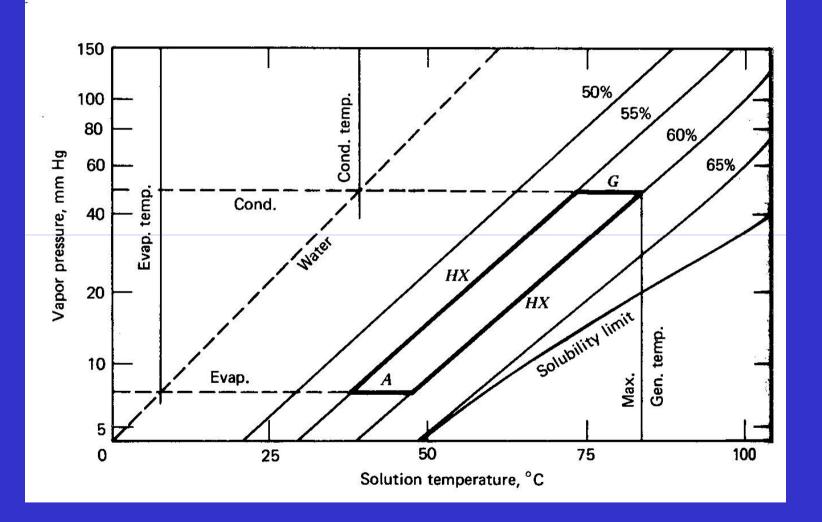
# Absorption Cooling (2)

#### Substances used

Absorbent	Refrigerant
LiBr	H <sub>2</sub> O
H <sub>2</sub> O	NH <sub>3</sub>

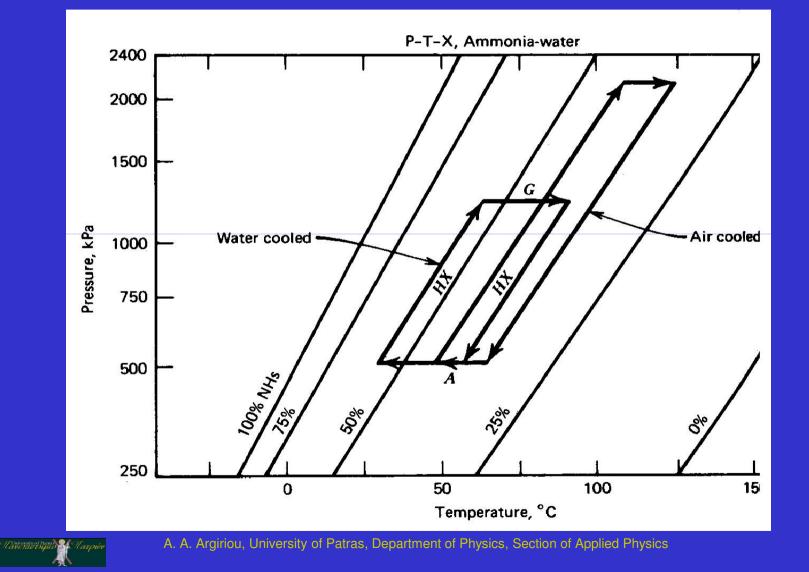


## Properties of LiBr – H<sub>2</sub>O





# Properties of $H_2O - NH_3$



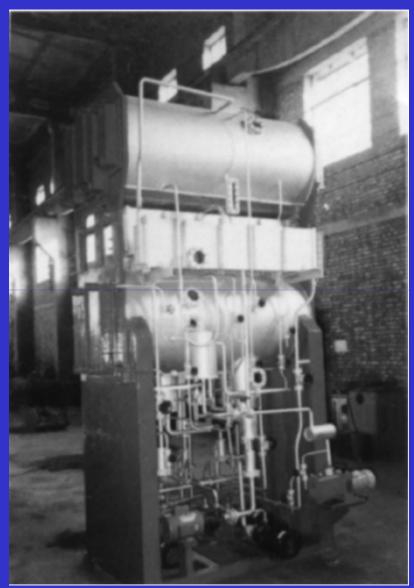
#### Real application – Solar collectors



Source: K. Sumathy, Z. C. Huang and Z. F. Li, Solar Energy, 2002, 72(2), 155-165



#### Absorption machine



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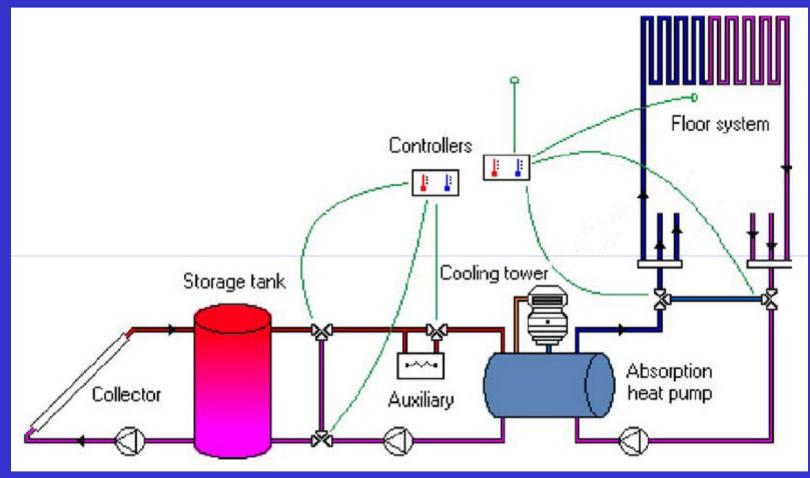


#### Single effect Yazaki machine (10 ton LiBr)





#### System combined to sub-floor exchanger





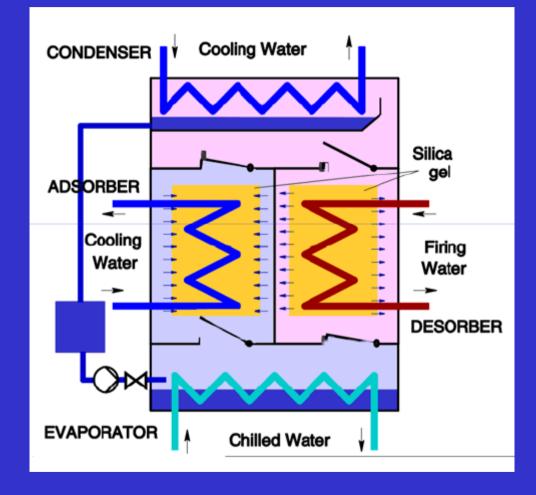
# Adsorption cooling

Adsorption is the use of solids for removing substances from gases and liquids The phenomenon is based on the preferential partitioning of substances from the gaseous or liquid phase onto the surface of a solid substrate.

The process is *reversible* 



# **Adsorption Cooling**





# Adsorption Cooling - Summary

The cycle is intermittent because production of cooling energy is not continuous: it occurs only during part of the cycle When there are two adsorbers in the unit, they can be operated separately and production of cooling energy can be quasi-continuous.

When all the energy required for heating the adsorber(s) is supplied by the heat source, the cycle is termed *single effect*.

Typically, for domestic refrigeration conditions, the COP of single effect adsorption cycles is of about 0.3-0.4.

When there are two adsorbers or more, other types of cycles can be designed.

In *double effect cycles* or in *cycles with heat regeneration*, some heat is internally recovered between the adsorbers, and that improves the COP.



# Adsorption cooling - Examples







## **Desiccant refrigeration**

#### Addresses the issue of thermal comfort by modifying the water vapor content in a space.

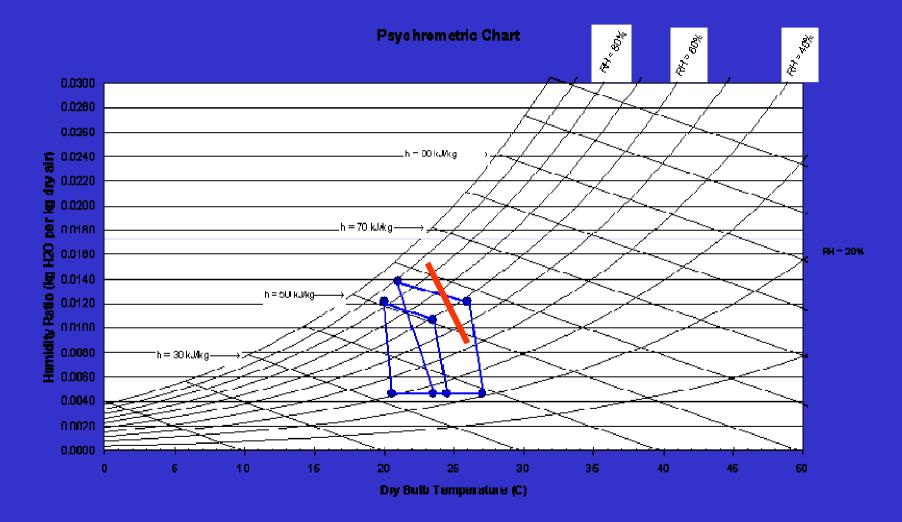
Decrease in bar wi indicates decrease	dth e in effect	Optimum zone	
Bacteria			
Viruses			
Fungi			
Mites			
Respiratory infections <sup>1</sup>			
Allergic rhinitis and asthma			
Chemical interactions			
Ozone production			
Percent relative hum	idity 10 20 30	40 50 60	70 80 90
<sup>1</sup> Insufficient data a	bove 50% R.H.		

#### **Effect of Room Relative Humidity on Selected Human Health Parameters**

Source: Sterling (1984) ASHRAE Transactions V. 90, Part 2.

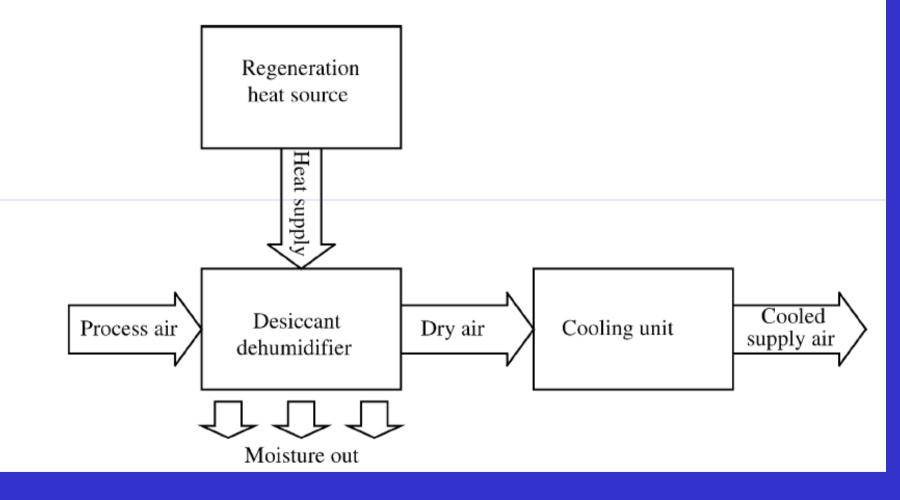


# Desiccant refrigeration principle





#### Desiccant refrigeration flow-chart





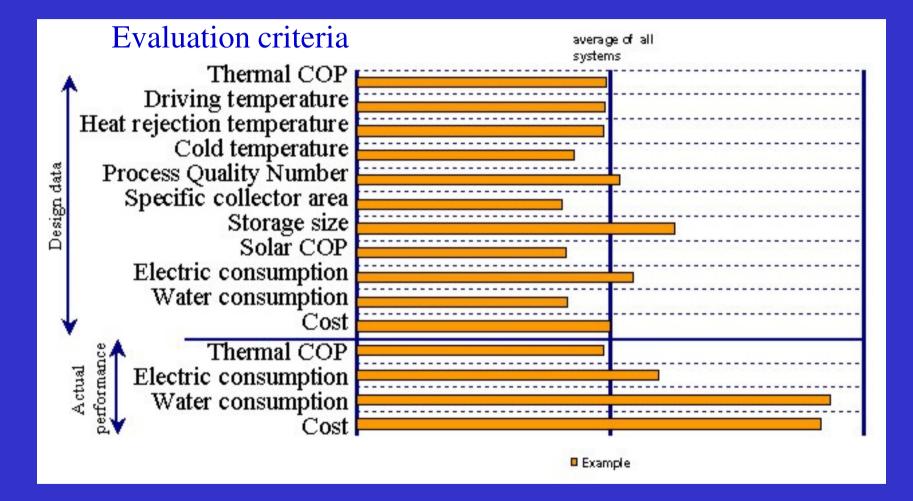
#### Solar cooling – Current status in Europe (source: EU SACE project)

Projects & applications identified and evaluated:

- 12 in Germany
- 2 in Austria
- 3 in Malta
- 1 in Croatia
- 5 in Greece
- 1 in Spain
- 1 in Kosovo
- 4 in Israel
- 15 from Cordis
- 10 IEA projects

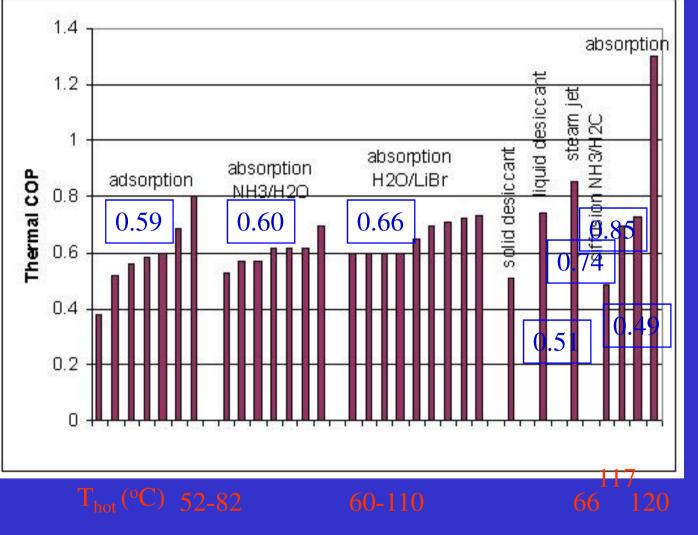


#### **Comparative assessment**





#### COP

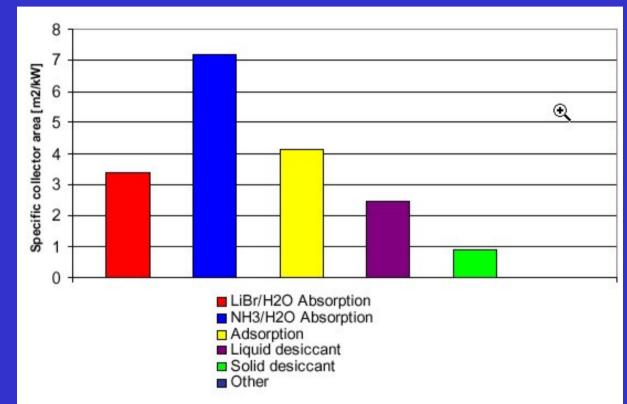


Διπλής βαθμίδας 1.3



#### Solar collectors used

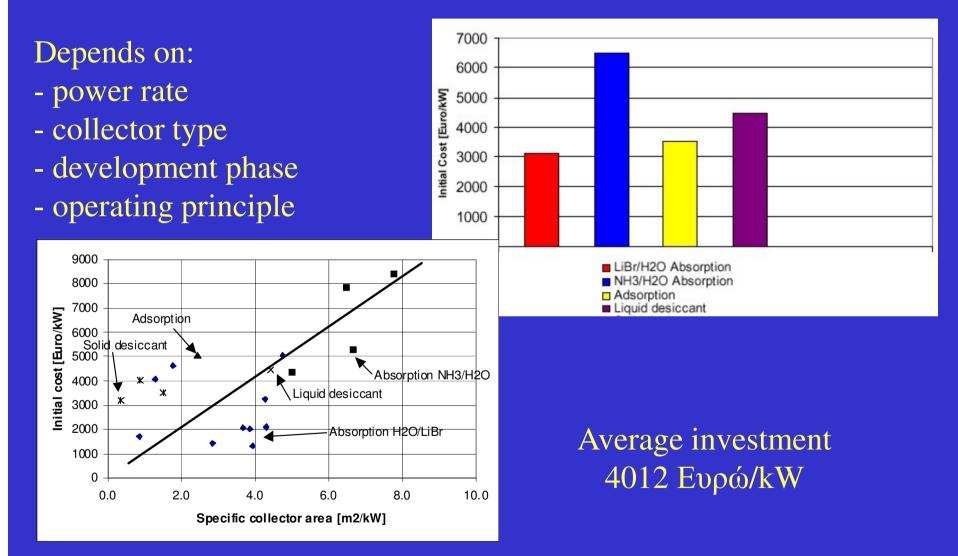
Flat-plated (63%) Vacuum tube (21%) Parabolic Fixed (10%) Moving (6%)



# Average specific collector area 3,6 m<sup>2</sup>/kW



#### Investment cost

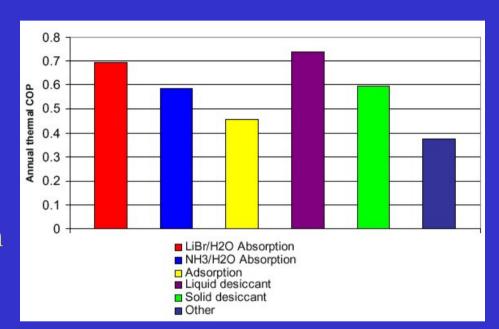




## Performance data

**Highest performance** LiBr / H<sub>2</sub>O systems

**Lowest performance** NH<sub>3</sub>/H<sub>2</sub>O diffusion system



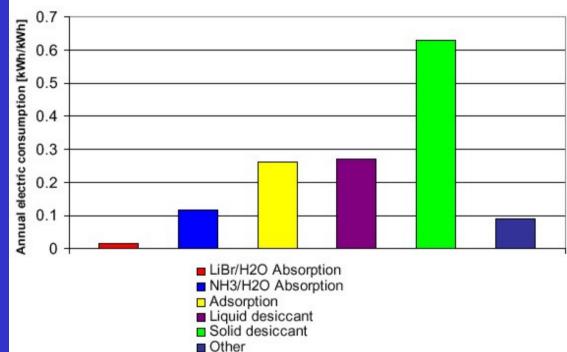
#### Average annual COP = 0.58



#### Consumption of auxiliary equipment

Lowest consumption: Absorption systems

*LiBr/H<sub>2</sub>O systems = 0.018 kWh/kWh* 



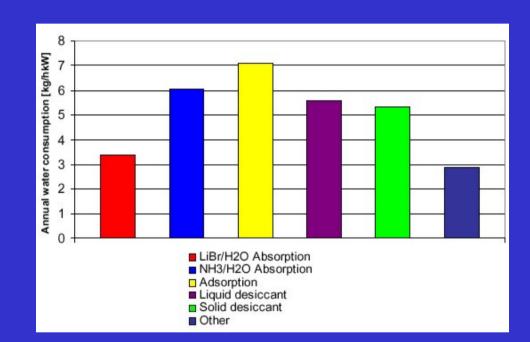
#### Mean annual electricity consumption of fans and pumps = 0.225 kWh/kWh



## Water consumption

Highest consumption Adsorption systems: 7.1 kg.h<sup>-1</sup>/kW

Majority of systems: 4-6 kg.h<sup>-1</sup>/kW



#### Mean annual water consumption = $5.3 \text{ kg.h}^{-1}/\text{kW}$



# Practical design guidelines

Detailed calculation of the energy budget of the application

- Energy savings depend on other energy sources used, i.e. gas boiler, auxiliary cooler, pumps, fans etc.
- Low COP coolers, require higher solar fraction and vice versa.
- Combined solar heating / cooling systems are more interesting financially



# Conclusions (1)

- Solar cooling is still in the development phase
- There are technological problems that need to be addressed mainly concerning the hydraulic circuit and the controllers
- Enough applications exist, but not enough performance data
- Reliable performance data and experience are available only from few systems



# Conclusions (2)

- Additional experience regarding the operation of real scale installations is necessary in order to develop model projects and solutions regarding network design and automatic control.
- Their market penetration requires further subsidies, <u>but</u>

only for systems that achieve important energy savings (e.g. >30%) with respect to conventional systems at a cost lower than a maximum price e.g.  $0,1 \in \text{per kWh of primary energy}$ .



#### Research priorities – LiBr systems

Increased performance and reduction of cost of solar collectors

Increased performance and reduction of cost of storage systems (e.g. thermochemical)

Development of low capacity absorption machines

Development of low capacity air-cooled absorption machines

Increased performance of the various heat transfer processes in the machine



#### Research priorities – NH<sub>3</sub> systems

Improved reliability, at low cost, independent control of the cooling medium

Improved pump reliability at low cost

Improved reliability of the fluid level sensors

Increased performance of the various heat transfer processes in the machine

Simplified system concepts

